

Phys 366
Lectures 14-15
Tensors

Who ordered tensors?

I. Inertia tensor.

Dynamics. Collection of particles with mass m_j position \vec{r}_j
 velocity $\vec{v}_j = \dot{\vec{r}}_j$, momentum $\vec{p}_j = m_j \vec{v}_j$. \vec{F}_j is the external
 force on particle j , and \vec{f}_{jk} is the internal force on
 particle j due to particle k . Newton's 3rd Law says
 that $\vec{f}_{jk} = -\vec{f}_{kj}$

Total momentum: $\vec{P} = \sum_j \vec{p}_j$

① $\frac{d\vec{P}}{dt} = \sum_j \dot{\vec{p}}_j = \sum_j \vec{F}_j + \underbrace{\sum_{j < k} \vec{f}_{jk}}_{= 0 \text{ by 3rd Law}} = \vec{F} = \left(\begin{array}{c} \text{total external} \\ \text{force} \end{array} \right)$
Newton's 2nd Law

Total angular momentum: $\vec{L} = \sum_j \vec{r}_j \times \vec{p}_j$

$\frac{d\vec{L}}{dt} = \sum_j \underbrace{\dot{\vec{r}}_j \times \vec{p}_j}_0 + \sum_j \vec{r}_j \times \dot{\vec{p}}_j = \sum_j \vec{r}_j \times \vec{F}_j + \underbrace{\sum_{j < k} \vec{r}_j \times \vec{f}_{jk}}_{\text{pairs}}$
Newton's 2nd Law
 $= \sum_{j < k} \vec{r}_j \times \vec{f}_{jk} + \sum_{k < j} \vec{r}_k \times \vec{f}_{kj}$
3rd Law $\rightarrow = \sum_{j < k} (\vec{r}_j - \vec{r}_k) \times \vec{f}_{jk}$
 $= 0$ for 3rd Law central forces

② $\frac{d\vec{L}}{dt} = \sum_j \vec{r}_j \times \vec{F}_j = \vec{N} = \left(\begin{array}{c} \text{total external} \\ \text{torque} \end{array} \right)$

Rigid bodies: particles maintain their relative orientation
 common displacement plus common rotation
 ① and ② are sufficient.

What you're used to: torque and angular momentum about one axis.

Dynamics

$$N_z = (\mathbf{r} \hat{\rho} \times \mathbf{F} \hat{\phi})_z = r F = \dot{L}_z$$

Kinematics

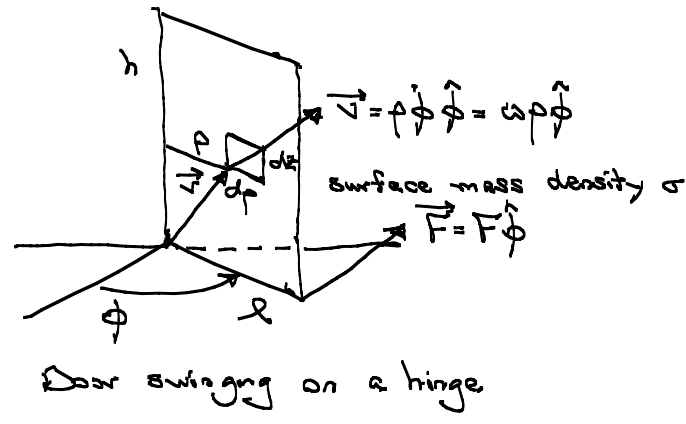
$$L_z = \int \sigma dp dz (\mathbf{r} \times \mathbf{v})_z$$

$$(\rho \hat{\phi} \times \omega \rho \hat{\phi})_z = \omega \rho^2$$

$$= \omega \sigma \int_0^l \rho^2 d\rho \int_0^h dz$$

$$= \frac{1}{3} \sigma l^3 h \omega$$

$$= \frac{1}{3} \frac{\sigma l h}{M} l^2 \omega$$



Equation of motion

$$Fl = N_z = \dot{L}_z = I \dot{\omega} = I \ddot{\phi}$$

$$L_z = \frac{1}{3} M l^2 \omega = I \omega$$

↑
moment of inertia

But the kinematics is generally more complicated than this. For an arbitrary rigid body, we have

$$\vec{p} = \int dm \vec{v} = \int dm \vec{\omega} \times \vec{r}$$

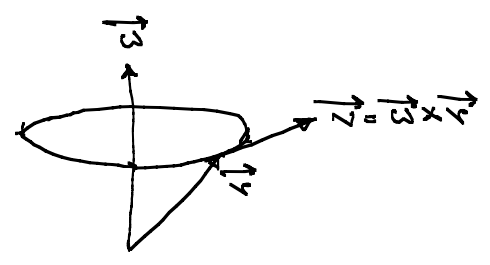
$$\vec{L} = \int dm \vec{r} \times \vec{v}$$

$$= \int \rho dV \vec{r} \times (\vec{\omega} \times \vec{r})$$

mass density

$$= \vec{\omega} (\vec{r} \cdot \vec{r}) - \vec{r} (\vec{r} \cdot \vec{\omega})$$

$$= r^2 \vec{\omega} - \vec{r} (\vec{r} \cdot \vec{\omega})$$



$$L_j = \int dV \rho (r^2 \omega_j - x_j x_k \omega_k) = \omega_k \int dV \rho (r^2 \delta_{jk} - x_j x_k) = I_{jk} \omega_k = L_j$$

$$I_{jj} = \int dV \rho (r^2 - x_j^2)$$

$$I_{jk} = - \int dV \rho x_j x_k$$

I_{jk} ← components of inertia tensor

The inertia tensor is a symmetric, rank-2 tensor. It is a linear operator on vectors to vectors.

$$\begin{pmatrix} L_1 \\ L_2 \\ L_3 \end{pmatrix} = \begin{pmatrix} I_{11} & I_{12} & I_{13} \\ I_{21} & I_{22} & I_{23} \\ I_{31} & I_{32} & I_{33} \end{pmatrix} \begin{pmatrix} \omega_1 \\ \omega_2 \\ \omega_3 \end{pmatrix}$$

Working abstractly, we have

unit tensor
Kronecker delta

$$\vec{L} = \int dV \rho (\underbrace{\vec{r} \otimes \vec{r}}_{\vec{I}} - \vec{r} \otimes \vec{r}) \cdot \vec{\omega}$$

$$\vec{I} = \int dV \rho (\vec{r} \otimes \vec{r} - \vec{r} \otimes \vec{r})$$

$$[\vec{I}] = M I^2$$

tensor OR direct product OR outer product

$$= \sum_{j,k} \hat{e}_j \otimes \hat{e}_k (\underbrace{\hat{e}_j \cdot \vec{I} \cdot \hat{e}_k}_{I_{jk}})$$

\vec{I} can also be thought of as a bilinear map from two vectors to a scalar: $\vec{v} \cdot \vec{I} \cdot \vec{w} = \langle \vec{v} | \vec{I} | \vec{w} \rangle$. Indeed, the kinetic energy of a rigid body is

$$\mathcal{K} = \frac{1}{2} \int dV \rho \underbrace{\vec{v} \cdot \vec{v}}_{\vec{I}} = \frac{1}{2} \vec{\omega} \cdot \int dV \rho \vec{r} \times (\vec{\omega} \times \vec{r}) = \frac{1}{2} \vec{\omega} \cdot \vec{I} \cdot \vec{\omega} = \frac{1}{2} \vec{\omega} \cdot [\vec{r} \times (\vec{\omega} \times \vec{r})]$$

$$\mathcal{K} = \frac{1}{2} \vec{\omega} \cdot \vec{I} \cdot \vec{\omega} = \frac{1}{2} \langle \vec{\omega} | \vec{I} | \vec{\omega} \rangle = (\omega_1 \ \omega_2 \ \omega_3) \begin{pmatrix} I_{11} & I_{12} & I_{13} \\ I_{21} & I_{22} & I_{23} \\ I_{31} & I_{32} & I_{33} \end{pmatrix} \begin{pmatrix} \omega_1 \\ \omega_2 \\ \omega_3 \end{pmatrix}$$

We'll work on these later.

Transformations: change of origin (parallel-axis theorem) and orthogonal transformations
Diagonalization, principal axes, and principal moments of inertia

$$I_{xx} = \int dV \rho (y^2 + z^2)$$

$$I_{yy} = \int dV \rho (x^2 + z^2)$$

$$I_{zz} = \int dV \rho (x^2 + y^2)$$

$$I_{xy} = - \int dV \rho xy = I_{yx}$$

$$I_{xz} = - \int dV \rho xz = I_{zx}$$

$$I_{yz} = - \int dV \rho yz = I_{zy}$$

Bra-ket version unit operator

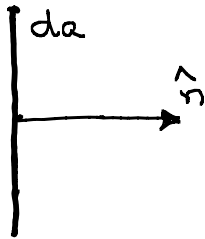
$$\vec{I} = \int \rho dV (\langle \vec{r} | \vec{r} \rangle \delta - |\vec{r}\rangle \langle \vec{r}|)$$

$$= \sum_{j,k} |e_j\rangle \langle e_j | \vec{I} | e_k\rangle \langle e_k|$$

I_{jk}

II. Stress tensor

The stress tensor $\overset{\leftrightarrow}{T}$ quantifies the flow of momentum across a surface. Since momentum flow is a force, $\overset{\leftrightarrow}{T}$ quantifies the force across surfaces.



$$\begin{aligned}
 \left(\begin{array}{l} \text{momentum that flows} \\ \text{across } da \text{ in time } dt \end{array} \right) &= \overset{\leftrightarrow}{T} \cdot \vec{da} dt = dt \sum_j \hat{e}_j (\hat{e}_j \cdot \overset{\leftrightarrow}{T} \cdot \hat{n}) da \\
 &\uparrow \\
 \text{Symmetric 2-tensor} & \\
 \left[\overset{\leftrightarrow}{T} \right] &= \frac{\text{force}}{\text{area}} = \frac{M}{T^2 L}
 \end{aligned}$$

$$= dt \sum_j \hat{e}_j T_{jk} n_k da$$

Symmetry guaranteed by angular-momentum conservation

$$\left(\begin{array}{l} \text{force left side exerts} \\ \text{on right side} \end{array} \right) = \left(\begin{array}{l} \text{rate of momentum} \\ \text{flow across } da \end{array} \right) = \overset{\leftrightarrow}{T} \cdot \vec{da}$$

↑ automatically 3rd Law

Force on a volume:

$$\begin{aligned}
 \vec{F} &= - \oint_S \overset{\leftrightarrow}{T} \cdot \vec{da} = - \int_V \nabla \cdot \overset{\leftrightarrow}{T} d\tau \\
 &\quad \downarrow \qquad \qquad \downarrow \\
 &\hat{e}_j T_{jk} \hat{e}_k \cdot \vec{da} \qquad \hat{e}_j T_{jk,k}
 \end{aligned}$$

$-\nabla \cdot \overset{\leftrightarrow}{T}$ is the force density

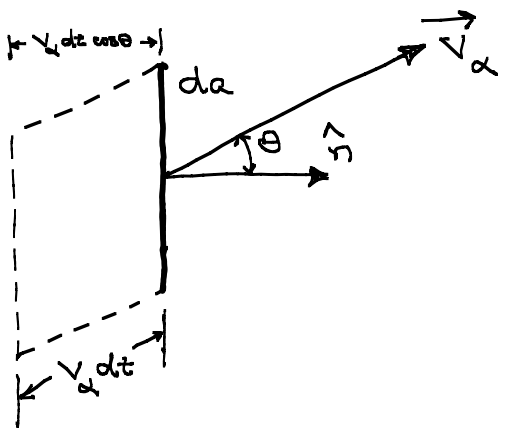
Extended example from transport theory.

What about particles with different masses, like the various molecules in air?

Particles carrying some property m , e.g., mass. At a particular point \vec{r} , divide the particles up into types. Particles of type α have number density n_α and have velocity \vec{v}_α . The overall number density is $n = \sum_\alpha n_\alpha$, and the mass density is $\rho = \sum_\alpha m n_\alpha = m n$. The average velocity is

$$\vec{v}(\vec{r}) = \langle \vec{v} \rangle = \frac{1}{n} \sum_\alpha n_\alpha \vec{v}_\alpha \longleftrightarrow \frac{1}{n} \int d^3v n(\vec{r}, \vec{v}) \vec{v}$$

Particles of type α :



(number of particles of type α that pass through da in dt) = $n_\alpha da v_\alpha dt \cos\theta$
 $= n_\alpha \vec{v}_\alpha \cdot d\vec{a} dt$

(mass current through da) = $\sum_\alpha m n_\alpha \vec{v}_\alpha \cdot d\vec{a}$
 $= \rho \vec{v} \cdot d\vec{a}$

mass current density

(mass current through S) = $\int_S \rho \vec{v} \cdot d\vec{a}$

Sign issues: sign of property direction of flow

Conservation of mass:

① Integral form

$$\frac{d}{dt} \int_V \rho d\tau = - \oint_S \rho \vec{v} \cdot d\vec{a} + \int_V R d\tau$$

$$\int_V \frac{\partial \rho}{\partial t} d\tau \quad \int_V \nabla \cdot (\rho \vec{v}) d\tau$$

source ($R > 0$) or sink ($R < 0$) rate

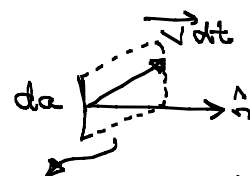
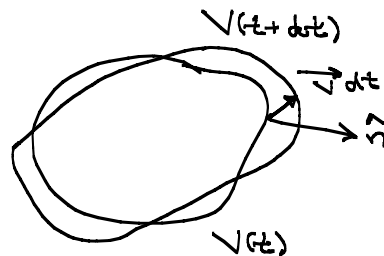
Differential form: $\frac{\partial \rho}{\partial t} = -\nabla \cdot (\rho \vec{v}) + R$

$R = 0$ is the conservation of mass

Integral form:

$$\frac{d}{dt} \int_{V(t)} \rho d\tau = \int_{V(t)} R d\tau$$

↑
comoving volume (follows the flow)



$$\text{(added volume)} = da \vec{v} \cdot \hat{n} dt = \vec{v} \cdot d\vec{a} dt$$

$$\frac{d}{dt} \int_{V(t)} d\tau = \frac{dV}{dt} = \oint_S \vec{v} \cdot d\vec{a} = \int_V \nabla \cdot \vec{v} d\tau$$

Differential form:

$$R \Delta V = \frac{d}{dt} (\rho \Delta V) = \frac{d\rho}{dt} \Delta V + \rho \frac{d\Delta V}{dt} = \Delta V \left(\frac{d\rho}{dt} + \rho \nabla \cdot \vec{v} \right)$$

↑
differential volume that follows the flow

Why is this the same?

$$\frac{d\rho}{dt} = \frac{\partial \rho}{\partial t} + \sum_j \frac{\partial \rho}{\partial x_j} \frac{dx_j}{dt} = \frac{\partial \rho}{\partial t} + \vec{v} \cdot \nabla \rho$$

$$\oint_S \frac{\partial \rho}{\partial t} d\tau = - \underbrace{\nabla \cdot \rho \vec{v}}_{-\nabla \cdot (\rho \vec{v})} + R$$

But what if the "property" carried by the particles is a vector and depends on the velocity, e.g., the momentum. Then the current is a vector, and the current density has to have two indices, one to dot into \$d\vec{a}\$ and the other to be the component of the momentum current. Thus the momentum current density is a rank-two tensor.

$$\begin{aligned} \left(\text{momentum that passes through } d\vec{a} \text{ in } dt \right) &= dt \sum_{\alpha} \vec{p}_{\alpha} v_{\alpha} \cdot d\vec{a} \\ &= dt \sum_{\alpha} m n_{\alpha} \vec{v}_{\alpha} (\vec{v}_{\alpha} \cdot d\vec{a}) \end{aligned}$$

$$\frac{1}{dt} \left(\text{momentum that passes through } \vec{da} \text{ in } dt \right) = \left(\sum_{\alpha} m n_{\alpha} \vec{v}_{\alpha} \otimes \vec{v}_{\alpha} \right) \cdot \vec{da}$$

force that $-\hat{n}$ side exerts on $+\hat{n}$ side of da

momentum current density $\underline{\underline{P}}$

force per area $\underline{\underline{P}}$

stress tensor

This force automatically satisfies Newton's 3rd Law

There is an important difference between the momentum current density and the mass current density, having to do with the average velocity $\vec{v} = \frac{1}{n} \sum_{\alpha} n_{\alpha} \vec{v}_{\alpha}$. Let's write

$$\vec{v}_{\alpha} = \vec{v} + \delta \vec{v}_{\alpha} \quad \sum_{\alpha} n_{\alpha} \delta \vec{v}_{\alpha} = 0$$

deviation from average flow

$$\begin{aligned} \sum_{\alpha} m n_{\alpha} \vec{v}_{\alpha} \otimes \vec{v}_{\alpha} &= \sum_{\alpha} m n_{\alpha} (\vec{v} + \delta \vec{v}_{\alpha}) \otimes (\vec{v} + \delta \vec{v}_{\alpha}) \\ &= \underbrace{\rho \vec{v} \otimes \vec{v}}_{\text{dynamic stress}} + \underbrace{3 \sum_{\alpha} n_{\alpha} \delta \vec{v}_{\alpha} \otimes \delta \vec{v}_{\alpha}}_{\text{stress tensor}} \\ &= \underbrace{m \sum_{\alpha} n_{\alpha} \delta \vec{v}_{\alpha} \otimes \vec{v}}_0 + m \vec{v} \otimes \underbrace{\sum_{\alpha} n_{\alpha} \delta \vec{v}_{\alpha}}_0 \end{aligned}$$

These are the dynamic stresses due to the bulk motion of the fluid (force exerted by the wind).

This is the stress tensor in the rest frame of the fluid, and it is what is conventionally called the stress tensor $\underline{\underline{T}}$. $\underline{\underline{T}}$ is manifestly symmetric, and for an ordinary fluid, which is isotropic, $\underline{\underline{T}}$ is a multiple of the unit tensor, with the multiple being the pressure p . Generally, $\underline{\underline{T}}$ is symmetric — otherwise, angular momentum is not conserved — but in more complicated fluids or solids, $\underline{\underline{T}}$ has off-diagonal terms called shear stresses.

Shear stresses correspond to momentum carried not by the particles, but by interactions between particles.

$$\sum_{\alpha} m n_{\alpha} \vec{v}_{\alpha} \otimes \vec{v}_{\alpha} = \rho \vec{v} \otimes \vec{v} + \vec{T}$$

Unlike the inertia tensor, the stress tensor is a tensor field.

$$\begin{aligned} \vec{T} &= m \sum_{\alpha} n_{\alpha} \vec{v}_{\alpha} \otimes \vec{v}_{\alpha} \longleftrightarrow m \int d^3v n(\vec{v}, \vec{r}) \vec{v} \otimes \vec{v} \\ &= \sum_{j,k} T_{jk}(\vec{r}) \hat{e}_j \otimes \hat{e}_k = \sum_{j,k} T_{jk}(\vec{r}) (\hat{e}_j \otimes \hat{e}_k) \\ &\longleftrightarrow \begin{pmatrix} T_{11} & T_{12} & T_{13} \\ T_{21} & T_{22} & T_{23} \\ T_{31} & T_{32} & T_{33} \end{pmatrix} = \begin{pmatrix} p & 0 & 0 \\ 0 & p & 0 \\ 0 & 0 & p \end{pmatrix} \longleftrightarrow p \vec{I} \end{aligned}$$

isotropic fluid

Fluid equations

- Ⓐ Mass conservation: $\frac{D\rho}{Dt} = -\nabla \cdot (\rho \vec{v})$
- Ⓑ Newton's end Law OR momentum flow
- Ⓒ Integral form:

$$\begin{aligned} \frac{d}{dt} \int_V \rho \vec{v} d\tau &= \int_V \vec{f} d\tau - \oint_S \rho \vec{v} \otimes \vec{v} \cdot d\vec{a} - \oint_S \vec{T} \cdot d\vec{a} \\ &\quad \text{external force density (source of momentum)} \\ &= \int_V \underbrace{\rho \vec{v} (\vec{v} \cdot d\vec{a})}_{\hat{e}_j \rho v_j \vec{v} \cdot d\vec{a}} = \int_V \underbrace{\nabla \cdot (\rho \vec{v} \otimes \vec{v})}_{\hat{e}_j \nabla \cdot (\rho v_j \vec{v}) = \hat{e}_j \frac{\partial}{\partial x_k} (\rho v_j v_k)} d\tau \\ &\quad \underbrace{\oint_S \vec{T} \cdot d\vec{a}}_{\hat{e}_j \frac{\partial T_{jk}}{\partial x_k} d\tau} \end{aligned}$$

If $\vec{T} = p \vec{I}$, then

$$\nabla \cdot \vec{T} = \hat{e}_j T_{jk,k} = \hat{e}_j \delta_{jk} p_{,k} = \hat{e}_j p_{,j} = \nabla p$$

Differential form:

$$\begin{aligned} \frac{\partial (\rho \vec{v})}{\partial t} &= \vec{f} - \underbrace{\nabla \cdot (\rho \vec{v} \otimes \vec{v})}_{\hat{e}_j \nabla \cdot (\rho v_j \vec{v}) = \hat{e}_j \nabla \cdot (\rho v_j \vec{v}) = \hat{e}_j \nabla \cdot (\rho v_j \vec{v})} - \nabla \cdot \vec{T} \\ &= \vec{f} - \nabla \cdot (\rho \vec{v} \otimes \vec{v}) - \nabla p \end{aligned}$$

Example: hydrostatic equilibrium $\frac{\partial \rho}{\partial t} = 0, \vec{v} = 0$

$$0 = \vec{f} - \nabla \cdot \vec{T} = -\rho g \hat{z} - \nabla p$$

Incompressible (water) vs.
Compressible (air)

The pressure gradient supports the fluid.

Buoyancy force.

② Integral form:

$$\frac{d}{dt} \int_{V(t)} \rho \vec{v} d\tau = \int \vec{f} d\tau - \oint_S \vec{T} \cdot d\vec{a}$$

↑
moving volume

Differential form:

$$\frac{d}{dt} (\rho \vec{v} \Delta V) = \vec{f} \Delta V - \nabla \cdot \vec{T} \Delta V$$

=

$$\frac{d(\rho \vec{v})}{dt} \Delta V + \rho \vec{v} \frac{d\Delta V}{dt} = \Delta V \left(\frac{d(\rho \vec{v})}{dt} + \rho \vec{v} \nabla \cdot \vec{v} \right)$$

$$\text{So } \frac{d(\rho \vec{v})}{dt} = \vec{f} - \rho \vec{v} \nabla \cdot \vec{v} - \nabla \cdot \vec{T}$$

$$\text{Equivalence: } \frac{d(\rho \vec{v})}{dt} = \frac{\partial(\rho \vec{v})}{\partial t} + \sum_j \frac{\partial(\rho \vec{v})}{\partial x_j} \frac{dx_j}{dt} = \frac{\partial(\rho \vec{v})}{\partial t} + \vec{v} \cdot \nabla (\rho \vec{v})$$

$$\text{So } \frac{\partial(\rho \vec{v})}{\partial t} = \vec{f} - \vec{v} \cdot \nabla (\rho \vec{v}) - \rho \vec{v} \nabla \cdot \vec{v} - \nabla \cdot \vec{T} \quad \checkmark$$